

Optimum Shape Design of a Radiant Oven by the Conjugate Gradient Method and a Grid Regularization Approach

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ABSTRACT

This study presents an optimization problem for shape design of a 2-D radiant enclosure with transparent medium and gray-diffuse surfaces. The aim of the design problem is to find the optimum geometry of a radiant enclosure from the knowledge of temperature and heat flux over some parts of boundary surface, namely the design surface. The solution of radiative heat transfer is based on the net radiation method, where the configuration factors are obtained by the Hottel's crossed-string approach by treating blockage and convex surfaces. The conjugate gradient method is used for minimization of an objective function, which is expressed as the sum of square residuals between estimated and desired heat fluxes over the design surface, and the sensitivity coefficients are calculated by the finite difference method. A regularization approach is proposed to numerically regularize the ill-ordered grids, which are commonly found during the iterative optimization process. Some example problems are presented to show the performance and accuracy of the method. The results show that the optimization procedure can successfully generate the optimum geometry of radiant enclosure.

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1. INTRODUCTION

Optimum design of radiant enclosures has gained considerable interest in semi conductor and food industries. The main goal of the design problem is to satisfy the uniform heat flux and temperature distributions over the surface of product, namely the design surface. This goal may be attained by regularization of the strengths or the locations of heat sources over the heater surface [1-4]. However, when setting a number of heaters with variable strengths or locations over the heater surface is difficult, expensive or non-practical, optimization of enclosure shape design may be the final solution. The shape optimization may also be combined with other optimization problems in order to achieve the best solution.

In recent years, shape optimization problems have received much attention in a number of engineering applications. Practical applications are found from the optimization of mechanical structures [5-7] to aerodynamic design [8, 9]. In the field of heat transfer,

many efforts have been made to optimize the design of fin profiles to produce the maximum heat loss for a specified fin volume. Malekzadeh et al. [10, 11] used a combination of the differential quadrature element method and the golden section search method for maximizing the heat dissipation for any given fin volume with combined convective-radiative heat transfer. Kundu and Das [12] developed a generalized methodology for the optimum design of longitudinal fins, pin fins, and radial fins with uniform volumetric heat generation using the variational method.

One of the powerful tools for optimization is the genetic algorithm. The genetic algorithm (GA) is an efficient searching method based on the mechanisms of natural selection for global optimization in complex systems. Fabri [13, 14] used the genetic algorithm for geometry design of a single fin in order to achieve the highest fin efficiency. The application of genetic algorithm for optimum shape design of fins with volumetric heat generation under the influences of free convection and radiation was presented by Azarkish et al. [15]. A few studies for optimum design of radiative enclosures were reported. Sarvari [16] applied the micro genetic algorithm for optimum design of a radiant oven

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by improving the location of a number of control points representing the B -spline curve of the radiant oven to achieve the desired goal. Because the GA searches from a population of points, the probability of the search getting trapped in a local minimum is limited. However, the large size of the population makes it very poor in terms of convergence performance so that the iterative procedure, even for micro genetic algorithm which uses a population of five chromosomes, is time consuming.

Another tool for optimization of thermal systems is the gradient-based methods such as the Newton method or the conjugate gradient method (CGM) [17]. The main drawback of gradient-based methods is that the result depends on the initial guess. If there is more than one local optimum in the problem, the result will depend on the choice of the starting point, and a global optimum cannot be guaranteed. Another drawback of gradient-based methods is that they depend on the existence of the known parameter with respect to the unknown variables can be found in a straightforward manner, these methods have a good performance, but in most cases it is difficult or extremely expensive to calculate these derivatives precisely. In spite of the above-mentioned drawbacks, the gradient-based methods are more efficient in convergence rate. This advantage compensates the drawbacks of the gradient-based methods, and hence even today these methods are widely used for inverse design of complex systems. For example Daun et al. [18, 19] used the gradient based optimization methods for geometry design of a radiant oven through the optimization of locations of a few control points for the B -spline curves representing the geometry. Lan et al. [20] reported an approach combining the curvilinear grid generation and conjugate gradient methods for shape design of heat conduction problems. They proposed a redistribution approach to regularize the ill-ordered grids which are commonly generated during the iterative optimization procedure.

In the present work, we attempt to find the optimum geometry of a radiant oven to produce the uniform thermal conditions over the design surface. A uniform heat flux is maintained over the heater surface, and the design surface is considered to be isothermal. Then, the aim of the design problem is to produce uniform heat flux distribution over the design surface through optimizing the profile of an adiabatic wall. The radiation transfer equation is solved by the net radiation method and the conjugate gradient method is used to optimize the locations of the nodal points which represent the profile of the adiabatic wall. An efficient regularization approach is used to remove the ill-ordered points which are generated during the iterative procedure and replacing them with new points in alignment. The performance of the optimization method is examined by some numerical experiments and the effects of the initial guess are discussed.

2. DESCRIPTION OF PROBLEM

Figure 1 shows a schematic shape of a radiant enclosure. The design surface is placed horizontally, and the heater surface has a deviation angle ϕ with the design surface. The nodal points shown in Figure 1 by $z_m(x_m, y_m)$ are the dynamic nodal points that represent the shape of the adiabatic wall. Both the design surface and the heater surface are diffuse-black, and the medium is transparent. A uniform heat input is maintained over the entire heater surface. The aim of the design problem is to optimize the geometry of the adiabatic wall by optimizing the locations of dynamic nodal points to produce a uniform heat flux distribution over the temperature-specified design surface.

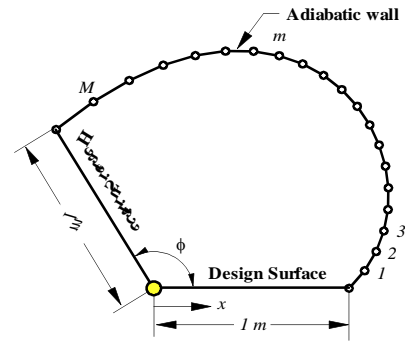


Figure 1. Schematic shape of the radiant oven and dynamic nodal points at the adiabatic surface

3. DIRECT PROBLEM

Consider a radiant enclosure with K discrete internal surfaces as shown in Figure 2a. The objective is to analyze the radiation exchange between the surfaces involving two types of boundary conditions; the surfaces with specified emissive power and the surfaces with specified heat flux.

The net radiation method is used to solve the radiation exchange in the radiant enclosure. In this method, the boundary is subdivided into surface elements. The equation of radiation exchange for surface elements with specified emissive power can be described by the following equation:

$$\sum_{j=1}^K [\delta_{kj} - (1 - \varepsilon_k) F_{k-j}] J_j = \varepsilon_k E_{b,k}, \quad 1 \leq k \leq K_1 \quad (1a)$$

whereas, the radiative transfer equation for other surface elements with specified heat flux is given by [21]:

$$\sum_{j=1}^K (\delta_{kj} - F_{k-j}) J_j = q_k, \quad K_1 + 1 \leq k \leq K \quad (1b)$$

where δ_{kj} is the Kronecker delta defined by:

$$\delta_{kj} = \begin{cases} 1 & k = j \\ 0 & k \neq j \end{cases} \quad (2)$$

here, J , q and E_b denote the radiosity, heat flux and the emissive power, respectively. The set of Eq. (1) is solved to calculate the outgoing heat fluxes, $J_j(t), j=1, \dots, K$, then the unknown boundary condition (emissive power or heat flux) is determined by the following equation:

$$\frac{1 - \epsilon_k}{\epsilon_k} q_k + J_k = E_{b,k}, \quad 1 \leq k \leq K \tag{3}$$

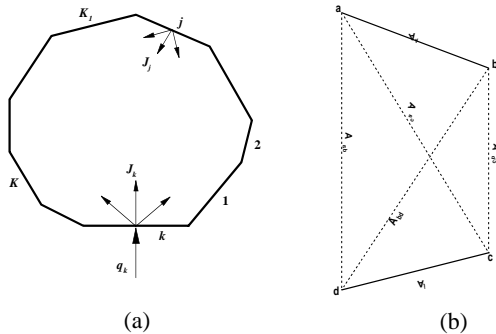


Figure 2. (a) Schematic shape of a radiant enclosure and (b) parameters in the Hottel's crossed-string method

The configuration factors are calculated by the Hottel's crossed-string method (see Figure 2b).

$$F_{k-j} = \frac{A_{ac} + A_{bd} - A_{ad} - A_{bc}}{2A_k} \tag{4}$$

Some considerations must be employed to treat the configuration factors of convex elements, and blockages. Figure 3a and b show these cases.

As shown in Figure 3a, the configuration factor of F_{k-j} between the element k and the convex element j must be vanished. However, the Hottel's crossed-string method do not predict the zero value for F_{k-j} . In order to treat this case, the view between two elements k and j must be checked before calculating the configuration factor. The configuration factor F_{k-j} exists if the value of cross product of a counterclockwise vector along the element k with a connecting vector from any point on element k to any point on element j be positive, or $|V_1 \times V_2| > 0$ (see Figure 3a).

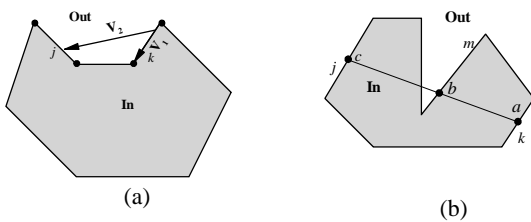


Figure 3. (a) Schematic shape of a radiant enclosure with convex surface elements and (b) schematic shape of a radiant enclosure with a blockage surface element

In Figure 3b the element m makes a blockage between elements k and j . In such cases, we draw a line from the midpoint on element k to the midpoint point on element j (line \overline{ac}). If this line intersects with element m at point b ($|\overline{ab}| < |\overline{ac}|$), the value of F_{k-j} is zero.

4. CONJUGATE GRADIENT METHOD

For the inverse problem considered here, the desired heat flux distribution over the design surface is available for the analysis, and the coordinates of the points constructing the adiabatic wall is regarded as unknowns. The desired and estimated heat flux distributions over the design surface may be expressed as vectors of discrete elemental values, such as:

$$\mathbf{q}_d = \{q_{d,n} \mid n=1, \dots, N\}^T, \tag{5}$$

$$\mathbf{q}_e = \{q_{e,n} \mid n=1, \dots, N\}^T$$

where N is the number of surface elements on the design surface. The vector of unknown coordinates of the nodal points constructing the adiabatic wall is expressed as:

$$\mathbf{Z} = \{z_m = (x_m, y_m) \mid m=1, \dots, M\}^T \tag{6}$$

where M is the number of nodal points over the adiabatic wall.

The solution is based on the minimization of the objective function given by:

$$G(\mathbf{Z}) = [\mathbf{q}_d - \mathbf{q}_e(\mathbf{Z})]^T [\mathbf{q}_d - \mathbf{q}_e(\mathbf{Z})] \tag{7}$$

The minimization procedure is performed using the conjugate gradient method. The CGM is an iterative procedure in which at each iteration a suitable step size, is taken along a direction of descent in order to minimize the objective function so that:

$$\mathbf{Z}^{\kappa+1} = \mathbf{Z}^{\kappa} + \beta^{\kappa} \mathbf{d}^{\kappa} \tag{8}$$

where the superscript κ is the iteration number. Here \mathbf{d}^{κ} is the direction of descent vector and β^{κ} is the search step size. The direction of descent can be determined as a conjugation of the gradient direction, ∇G , and the direction of descent from the previous iteration as follows:

$$\mathbf{d}^{\kappa} = \nabla G(\mathbf{Z}) + \alpha^{\kappa} \mathbf{d}^{\kappa-1} \tag{9}$$

where α^{κ} is the conjugation coefficient given by:

$$\alpha^{\kappa} = \frac{\nabla G(\mathbf{Z}) \nabla G^T(\mathbf{Z})}{\nabla G(\mathbf{Z}) \nabla G^T(\mathbf{Z})} \quad \text{with } \alpha^0 = 0 \tag{10}$$

The gradient direction is determined by differentiating

Eq. (7) with respect to the unknown parameter:

$$\nabla G(\mathbf{Z}) = -2\mathbf{S}^T [\mathbf{q}_d^\kappa - \mathbf{q}_c^\kappa(\mathbf{Z})] \quad (11)$$

where \mathbf{S} is the sensitivity (or Jacobian) matrix. The elements of the sensitivity matrix are calculated by the finite difference approach such as:

$$S_{n(2m-1)}^\kappa = \frac{\partial q_{e,n}^\kappa}{\partial x_m} \cong \frac{q_{e,n}^\kappa(x_1, \dots, x_m + \xi x_m, \dots) - q_{e,n}^\kappa(x_1, \dots, x_m, \dots)}{\xi x_m} \quad (12a)$$

$$S_{n(2m)}^\kappa = \frac{\partial q_{e,n}^\kappa}{\partial y_m} \cong \frac{q_{e,n}^\kappa(y_1, \dots, y_m + \xi y_m, \dots) - q_{e,n}^\kappa(y_1, \dots, y_m, \dots)}{\xi y_m} \quad (12b)$$

The estimated heat fluxes can be linearized with a Taylor series expansion and then minimization with respect to step size is performed to yield the following expression for the step size:

$$\beta^\kappa = \frac{[\mathbf{S}^\kappa \mathbf{d}^\kappa]^T [\mathbf{q}_d - \mathbf{q}_c^\kappa(\mathbf{Z})]}{[\mathbf{S}^\kappa \mathbf{d}^\kappa]^T [\mathbf{S}^\kappa \mathbf{d}^\kappa]} \quad (13)$$

The iterative procedure is stopped when the objective function becomes less than a pre-defined small value or the number of iterations reaches to a specified value.

5. IMPROVEMENT OF THE ILL-ORDERED POINTS

During the iterative solution procedure of the shape optimization, it is possible to generate unphysical shapes because of ill-ordered nodal points. The ill-ordered nodal points at the adiabatic wall represent abrupt changes in shape or unphysical profiles. Some various types of ill-ordered patterns, including out of alignment, twisted, and crossed grids are shown in Figures 4a-c, respectively.

A regularization approach is used to regularize the ill-ordered grids for the subsequent iterations, where the location of the nodal points at the adiabatic wall can be adjusted in order to have a smooth and continuous profile. The regularization approach considered in this study includes three steps: (i) detection, (ii) alignment, and (iii) redistribution. Next section represents the detailed description of the regularization approach.

5.1. Detection of Ill-ordered Grid To recognize the ill-ordered nodal points due to the out of alignment and twisted grids (see Figures. 4a and b), a deviation coefficient is defined as:

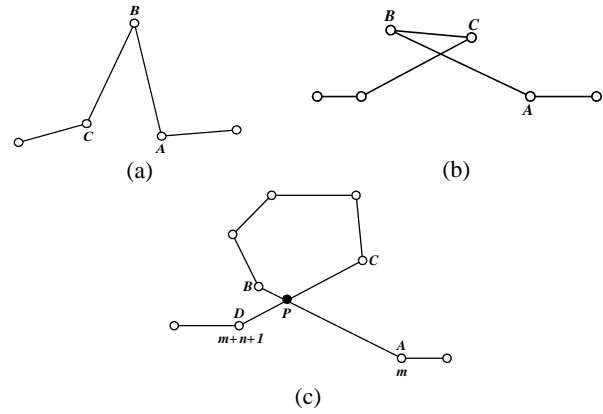


Figure 4. Ill-ordered grids (a) out of alignment grid, (b) twisted grid and (c) crossed grid

$$\frac{|\overline{AB}| + |\overline{BC}| - |\overline{AC}|}{|\overline{AC}|} \geq \alpha \quad (14)$$

where $|\overline{AB}|$ denotes the length of element \overline{AB} . The value of α is selected by trial and error process. According to [20], the deviation coefficient is set to 0.3 in this study. A larger α means a higher tolerance level for the ill-ordered points. The twisted and out of alignment grids may be detected using Eq. (14). These detected points must be replaced by new nodal points.

For detection of crossed grid, first the intersections of all non-successive elements are determined. Let the point P represents the intersection of the element \overline{AB} with the element \overline{CD} (see Figure 4c), then the location of the intersection point is given by:

$$x_P = \frac{b_{AB} - b_{CD}}{a_{CD} - a_{AB}} \quad (15a)$$

$$y_P = \frac{a_{AB}b_{CD} - a_{CD}b_{AB}}{a_{CD} - a_{AB}} \quad (15b)$$

where a and b denote the slope and the intercept of line, respectively. Now, the intersection point lies within both elements \overline{AB} and \overline{CD} if $|\overline{PA}| + |\overline{PB}| \leq |\overline{AB}|$ and $|\overline{PC}| + |\overline{PD}| \leq |\overline{CD}|$.

5.2. Alignment Let the point B be the point detected by Eq. (14) which lies between two nodal points A and C (see Figure 4a and b). This ill-ordered point is to be removed, and a new nodal point in alignment is created to replace it. The new nodal point is assigned to be located as:

$$z_B^* = (z_A + z_C)/2 \quad (16)$$

where z may be x or y , and superscript $*$ denotes the new value. Figure 5a and b show the shape of grids after alignment.

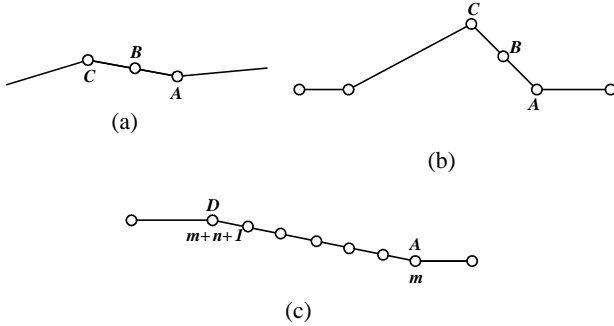


Figure 5. Ill-ordered grids (a) out of alignment grid, (b) twisted grid and (c) crossed grid, after alignment

For the case of a crossed grid, as depicted in Figure 4c, the starting point of the first crossed element is connected to the ending point of the second crossed element, and the ill-ordered nodal points are distributed equally over the connecting line. Hence, the locations of ill-ordered points are modified according to the following equation:

$$z_{m+i}^* = z_m + i \left(\frac{z_{m+n+1} - z_m}{n+1} \right), \quad i = 1, \dots, n \quad (17)$$

where n is the number of ill-ordered nodal points. The modified shape of crossed grid is shown in Figure 5c.

In addition to existence of a crossed grid at the adiabatic wall, during the iterative procedure it is possible that the dynamic elements at the adiabatic wall profile cross the static elements at either the design surface or the heater surface. Figure 6a shows such a case where the adiabatic wall profile crossed the design surface. In order to remove the ill-ordered points, the nodal point just before the first cross is connected to the nodal point just after the last cross at the adiabatic wall profile, and the locations of ill-ordered nodal points are updated based on Eq. (17). Figure 6b shows the modified grid after alignment.

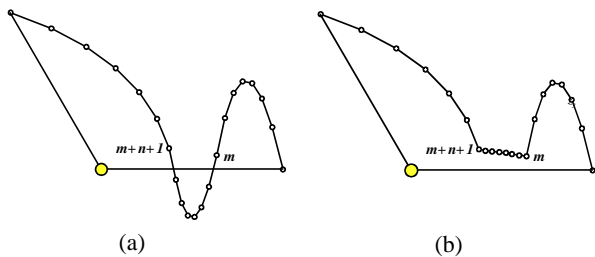


Figure 6. Crossing the adiabatic wall with the design surface (a) before alignment, and (b) after alignment

5.3. Redistribution After the alignment step, the nodal points may be distributed in a non-uniform manner, and thus the rate of convergence for iterative procedure may be decreased accordingly. In order to achieve a near uniform grid, we use a redistribution approach. First, the average length of elements is calculated by:

$$\ell_{ave} = \sum_{m=1}^{M+1} \ell_m / (M+1) \quad (18)$$

where ℓ_m is the length of the m -th element restricted to dynamic nodal points $m-1$ and m . Then, the number of segments over the m -th element is determined by:

$$S_m = \text{NINT} \left[\left(\ell_m / \ell_{ave} \right) \psi \right], \quad m = 1, \dots, M+1 \quad (19)$$

here the operator “NINT” rounds its argument to the nearest integer number, and ψ is the coefficient of segmentation. The value of ψ is selected based on trial and error process. The value of ψ is set to be 10 in this study. Finally, the location of artificial nodal point i at the m -th element is calculated by:

$$z_{m,i} = z_{m-1} + i \times \left(\frac{z_m - z_{m-1}}{S_m} \right), \quad i = 1, \dots, S_m, \quad m = 1, \dots, M+1 \quad (20)$$

Therefore, the new number of artificial sub-elements which must be laid between two dynamic nodal points is given by:

$$\delta = \text{NINT} \left[\sum_{m=1}^{M+1} S_m / (M+1) \right] \quad (21)$$

By defining

$$w_p = z_{i,j}, \quad p = 1, \dots, \sum_{m=1}^{M+1} S_m, \quad i = 1, \dots, M+1, \quad j = 1, \dots, S_m \quad (22)$$

The new location of the m -th dynamic nodal point becomes:

$$z_m^* = w_{m \times \delta}, \quad m = 1, \dots, M \quad (23)$$

with $z_0^* = z_0$ and $z_{M+1}^* = z_{M+1}$.

Figure 7 shows the locations of the dynamic nodal points at the adiabatic wall before and after redistribution.

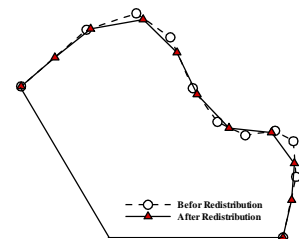


Figure 7. The schematic shape of the adiabatic wall profile before and after redistribution

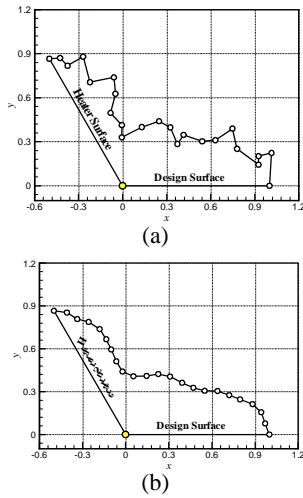


Figure 8. The geometry of the radiant oven for iteration 13 (a) before regularization and (b) after regularization

Typical regularization procedures for different number of iterations are shown in Figures 8-10. As seen, although the dynamic nodal points are seriously twisted or out of alignment at the start of the regularization procedure, a well-ordered grid distribution can be obtained after few iterations. It is also observed that the uniformity of the element length could be improved through the redistribution.

5.4. The Stopping Criterion The stopping criterion for the iterative procedure of nodal regularization is given by:

$$\sum_{m=1}^M (z_m^* - z_m)^2 \leq \xi \quad (24)$$

where ξ is a pre-defined small value.

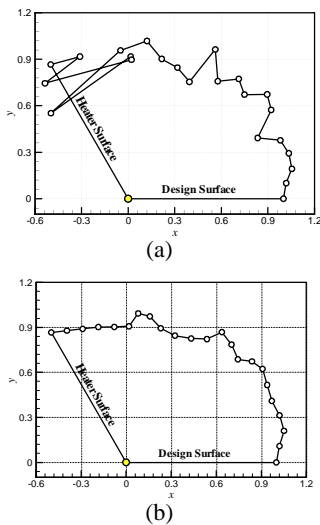


Figure 9. The geometry of the radiant oven for iteration 82 (a) before regularization and (b) after regularization

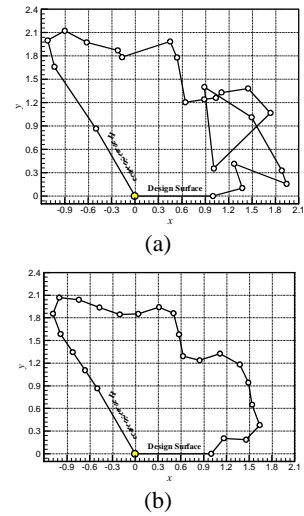


Figure 10. The geometry of the radiant oven for iteration 471 (a) before regularization and (b) after regularization

6. GRID REFINEMENT FOR THE DIRECT PROBLEM

The surface elements of the adiabatic wall may be stretched during the regularization procedure, in such a way that the direct problem of solving the radiative transfer equation is influenced by the discretization error. In order to reduce the discretization error, the surface grid at the adiabatic wall must be refined in each iteration. The refinement approach used in this study is based on the segmentation of surface elements into a number of sub-elements so that the number of sub-elements can be found by:

$$k_m = \text{NINT}[\ell_m / \ell_{ave}] , m = 1, \dots, M \quad (25)$$

7. OVERALL COMPUTATIONAL ALGORITHM

The overall computational algorithm for shape optimization of a radiant enclosure is shown in Figure 11. The flowchart shows the overall solution procedure containing the direct problem, conjugate gradient method and the nodal regularization approach.

8. RESULTS AND DISCUSSION

The optimization technique based on the CGM and the regularization approach is now demonstrated by applying it for design of a radiant oven as depicted in Figure 1, for different values of ϕ angle. All the surfaces are considered to be diffuse-black and the medium is transparent. A uniform heat input, $q_h = 1 \text{ W/m}^2$ is posed over the heater surface. The aim of the design problem is to optimize the geometry of the adiabatic wall to produce a uniform heat flux

The values of maximum relative error, the root mean square error, the objective function, and the CPU time per iteration for cases with different values of angles of deviation are shown in Table 1. As seen in Table 1, although the maximum relative error for some cases is large, but the values of root mean square error for almost all cases are acceptable.

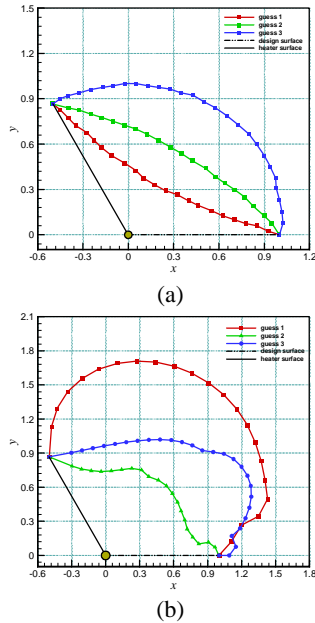


Figure 12. (a) Three initial guesses for adiabatic wall profile, (b) the obtained profiles of the adiabatic wall for three initial guesses, $\phi = 120^\circ$

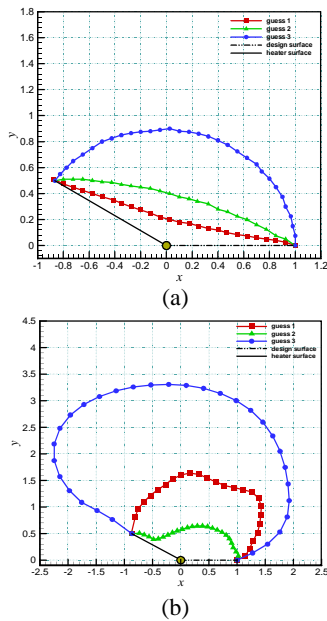


Figure 13. (a) Three initial guesses for adiabatic wall profile, (b) the obtained profiles of the adiabatic wall for three initial guesses, $\phi = 150^\circ$

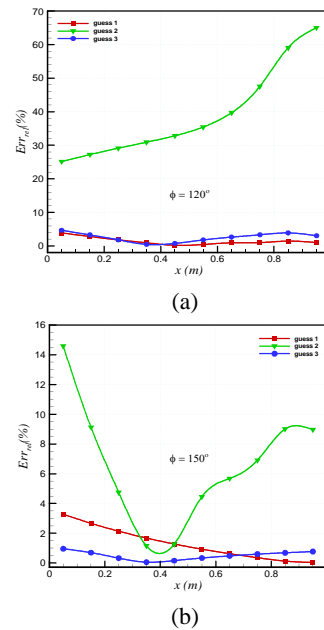


Figure 14. The distribution of relative error of estimated heat flux over the design surface for (a) $\phi = 120^\circ$ and (b) $\phi = 150^\circ$

TABLE 1. The maximum relative error, the root mean square error, the objective function and the CPU time per iteration for cases with different angles of deviation and different initial guesses

ϕ	Guess	$Er_{rel,max}$ (%)	Er_{rms} (%)	G	CPU time (s)
120°	1	3.87	0.018	0.003	1.89
	2	65.03	0.413	0.171	1.13
	3	4.66	0.029	0.008	1.78
150°	1	3.28	0.017	0.003	2.78
	2	14.58	0.0763	0.058	2.54
	3	0.96	0.006	0.003	29.33

Moreover, the CPU time is highly dependent to the number of nodal points, and the number of iterations for regularization of the locations of nodal points.

9. CASE STUDY

In order to check the performance of shape optimization method for reconstruction of a practical enclosure, we now consider a design problem of an oven with a 1-m design surface which is located horizontally at a distance of $h = 1\text{ m}$ from a cylindrical heater ($d = 0.2\text{ m}$), as depicted in Figure 15. The design surface is a diffuse-gray surface with an emissivity of $\epsilon_d = 0.7$ which has a specified uniform emissive power of $E_d = 1\text{ W/m}^2$. The cylindrical heater is diffuse-black and produces a constant heat flux of $q_h = 1/(\pi d)\text{ W/m}^2$. All other surfaces are adiabatic. The goal of the design problem is to identify the enclosure configuration to produce the uniform desired heat flux of $q_d = -1\text{ W/m}^2$ over the design surface.

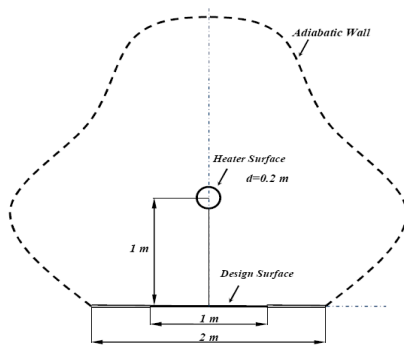


Figure 15. Schematic shape of an enclosure consisting of a cylindrical heater and a design surface located horizontally

Because of symmetry, only half the points constructing the adiabatic wall are considered to be optimized during the iterative regularization approach. The final shape is indicated in Figure 16. The values of objective function, maximum relative error, root mean square error, and the CPU time per each iteration are reported in Table 2. Figure 17 shows the distribution of relative error over the design surface. As seen, the desired uniform heat flux distribution over the design surface is well satisfied by optimization of the adiabatic wall.

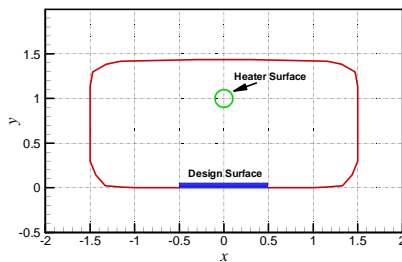


Figure 16. The obtained geometry for the case shown in Figure 15

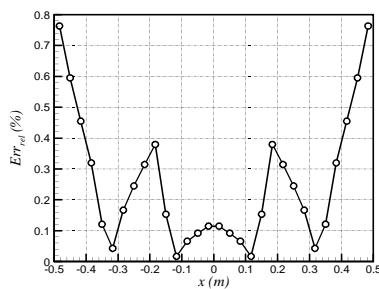


Figure 17. The distribution of relative error of estimated heat flux over the design surface for the geometry shown in Figure 16

TABLE 2. The maximum relative error, the root mean square error, the objective function, and the CPU time per iteration for the case described by Figure 15

$Er_{rel,max}$ (%)	Er_{rms} (%)	G	CPU time (s)
0.76	0.003	0.0003	14.74

10. CONCLUSION

This article overviewed a gradient-based approach for geometric design of a radiant oven. The primary objective of the work was to find the optimal shape of a radiant enclosure to satisfy the uniform heat flux distribution over the temperature-specified design surface. The radiative transfer equation in the radiant enclosure with diffuse-gray walls, containing a transparent medium was solved by the net radiation method and the view factors were found by the Hottel’s crossed string method. The optimal shape of the enclosure was found by optimizing the locations of dynamic nodal points at the adiabatic wall. The optimization problem was solved by the conjugate gradient method. A regularization approach was posed to the nodal points in order to remove the ill-ordered nodal points and generate a smooth profile. The results show that the optimum shape can be successfully generated by the present method in an acceptable range of error. However, the final solution is highly dependent to the initial guess.

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Optimum Shape Design of a Radiant Oven by the Conjugate Gradient Method and a Grid Regularization Approach

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در این مطالعه یک مسئله بهینه سازی جهت طراحی یک محفظه تابشی با محیط شفاف و سطوح پخشی-خاکستری ارائه شده است. هدف مسئله طراحی یافتن هندسه بهینه یک محفظه تابشی با استفاده از اطلاعات دما و شار حرارتی روی قسمت هایی از سطح مرزی که سطح طراحی نام دارد می باشد. حل انتقال حرارت تابشی براساس روش تابش خالص است که در آن ضرایب شکل با استفاده از روش تارهای متقاطع هاتل با در نظر گرفتن موانع و سطوح محدب بدست می آیند. روش شیب مزدوج بمنظور کمینه کردن یک تابع هدف، که بصورت مجموع مربعات اختلاف بین شارهای حرارتی مطلوب و ارزیابی شده روی سطح طراحی تعریف می شود، مورد استفاده قرار می گیرد. ضرایب حساسیت با استفاده از روش اختلاف محدود محاسبه می شوند. یک راهکار تنظیمی جهت تنظیم گره های درهم ریخته ارائه شده است که در یک فرآیند تکراری عمل می کند. بمنظور نمایش عملکرد و دقت روش مثال هایی ارائه گردیده است. نتایج نشان می دهد که هندسه بهینه محفظه تابشی بگونه ای موفقیت آمیز می تواند تولید شود.

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